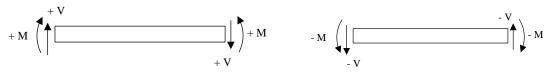
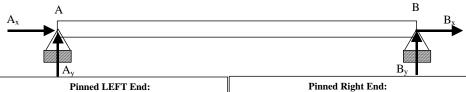
Some Guidelines for Constructing Shear Force and Bending Moment Diagrams

SIGN CONVENTIONS



BOUNDARY CONDITIONS



Reaction force, Ax is unknown. Reaction force, A_y is unknown. V (shear force) = A_y (reaction force) M = 0 (unless there is an applied moment at this point) Deflection, y = 0Slope, θ is unknown

Reaction force, Bx is unknown. Reaction force, By is unknown. V (shear force) = $-\mathbf{B}_{\mathbf{v}}$ (reaction force) M=0 (unless there is an applied moment at this point) Deflection, y = 0Slope, θ is unknown

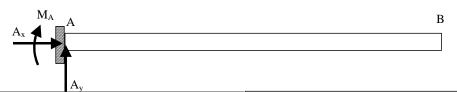


Pinned LEFT End:

Reaction force, A_x is unknown. Reaction force, A_y is unknown. V (shear force) = A_y (reaction force) M = 0 (unless there is an applied moment at this point) Deflection, y = 0Slope, θ is unknown

Roller Support at Right End:

Reaction force, $B_x = 0$. Reaction force, By is unknown. V (shear force) = $-\mathbf{B}_{\mathbf{y}}$ (reaction force) M = 0 (unless there is an applied moment at this point) Deflection, y = 0Slope, θ is unknown



Clamped End:

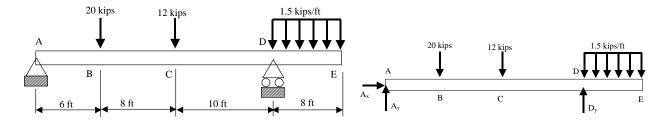
Reaction force, A_x is unknown. Reaction force, A_v is unknown. Reaction moment, MA is unknown V (internal shear force) = A_y (reaction force) M (internal bending moment) = M_A Deflection, y = 0Slope, $\theta = 0$

Free End:

V (shear force) = 0 (unless there is an applied point force at this end) M = 0 (unless there is an applied moment at this point) Deflection, y is unknown Slope, $\boldsymbol{\theta}$ is unknown

Example:

Draw the shear force and bending moment diagram for the following beam:

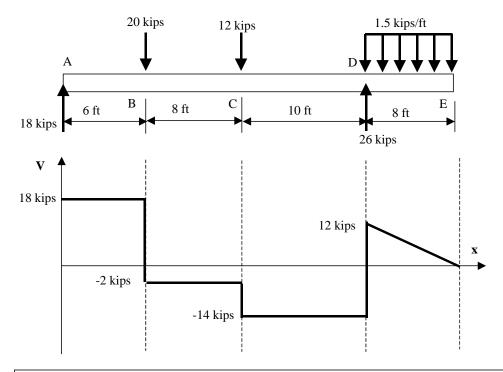


Step 1: Find the reaction forces at A and D.

Draw F.B.D.:

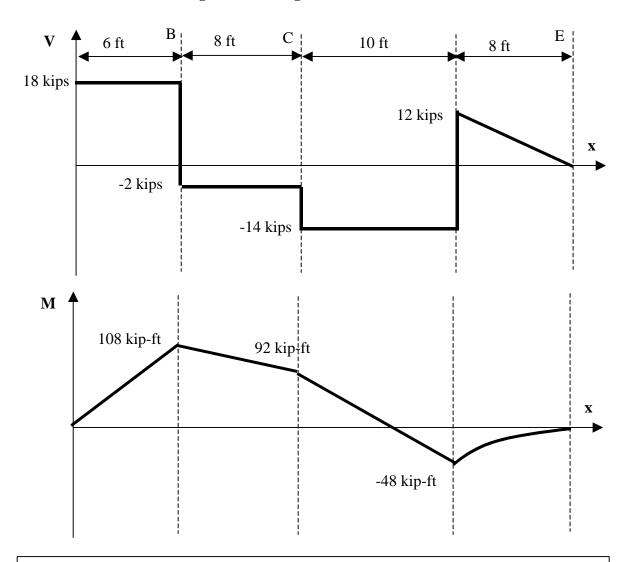
$$\sum F_y = 0$$
, $A_y + D_y - 20 \text{ kips} - 12 \text{ kips} - (1.5 \text{ kips/ft})(8 \text{ ft}) = 0$, $A_y = 18 \text{ kips}$

Construction of the shear force diagram



- a) $V = A_v = 18$ kips at point A.
- b) The shear force is constant until there is another load applied at B.
- c) The shear force decreases by 20 kips to -2 kips at B because of the applied 20 kip force in the negative y direction.
- d) The shear force is constant until there is another load applied at C.
- e) The shear force decreases by 12 kips to -14 kips at C because of the applied 12 kip force in the negative y direction.
- f) The shear force is constant until there is another load applied at D.
- g) The shear force increases by 26 kips to 12 kips at D because of the 26 kip reaction force in the positive y direction.
- h) The shear force decreases linearly from D to E because there is a constant applied load in the negative y-direction.
- The change in shear force from D to E is equal to the area under the load curve between D and E, -12 kips, $[A_{DE} = (-1.5 \text{ kips/ft})(8 \text{ ft}) = -12 \text{ kips}]$
- j) The shear force at E = 0 as expected by inspection of the boundary conditions.

Construction of the Bending Moment diagram



- a) M = 0 at point A because it is a pinned end with no applied bending moment.
- b) $M_B = M_a +$ (the area under the shear force diagram between A and B.)
- c) $M_B = 0 + (18 \text{ kips})(6 \text{ ft}) = 108 \text{ kip-ft}$
- d) $M_C = M_B + \text{(the area under the shear force diagram between B and C.)}$
- e) $M_C = 108 \text{ kip-ft} (2 \text{ kips})(8 \text{ ft}) = 92 \text{ kip-ft}$
- f) $M_D = M_C + \text{(the area under the shear force diagram between C and D.)}$
- g) $M_D = 92 \text{ kip-ft} (14 \text{ kips})(10 \text{ ft}) = -48 \text{ kip-ft}$
- h) $M_E = M_D + \text{(the area under the shear force diagram between D and E.)}$
- i) $M_E = -48 \text{ kip-ft} + \frac{1}{2} (12 \text{ kips})(8 \text{ ft}) = 0 \text{ kip-ft (as expected)}$